Lecture 29

Generalized Forces

$$p_i = \frac{\partial \mathcal{L}}{\partial \dot{q}_i} \implies \dot{p}_i = \frac{\partial \mathcal{L}}{\partial q_i}$$

where p_i is the generalized momentum and $\partial \mathcal{L}/\partial q_i$ is the general force. The second equation is Lagrange's equation.

Particles in 1D:
$$\mathcal{L} = \frac{1}{2}m\dot{x}^2 - V(x)$$

 $F_x = \frac{\partial \mathcal{L}}{\partial x} = -\frac{\partial V}{\partial x}$

Pendulum:



$$\mathcal{L} = \frac{1}{2}m(a\dot{\theta})^2 - mga(1 - \cos\theta))$$

$$F_{\theta} = \frac{\partial \mathcal{L}}{\partial \theta} = -mga\sin\theta = \text{Torque}$$

$$p_{\theta} = \frac{\partial \mathcal{L}}{\partial \dot{\theta}} = ma^2 \dot{\theta} = I\omega = \text{Angular Momentum}$$

$$\dot{p}_{\theta} = F_{\theta} \implies I\alpha = \tau$$

Equilibrium and Stability



- x_1 and x_2 are equilibrium points since $\partial V / \partial x|_{x=x_1,x_2} = 0$
- x_1 is unstable since $\partial^2 V / \partial x^2|_{x=x_1} < 0$
- x_2 is stable since $\partial^2 V / \partial x^2|_{x=x_2} > 0$

We wish to generalize to a single coordinate q:

$$V(q) = V_0 + V'_0 q + \frac{1}{2} V''_0 q^2 + \dots + \frac{1}{n!} V_0^{(n)} q^n + \dots$$

where $V_0 = V(q = 0)$ and $V'_0 = dV/dq|_{q=0}$ and so on. If q = 0 is a point of equilibrium, $V'_0 = 0$. (If it is not, we can define $q' = q - q_0$ where q_0 is the equilibrium point and repeat the argument).

Consequently, for small q we may approximate (Set $V_0 = 0$)

$$V(q)\approx \frac{1}{2}V_0''q^2$$

Provided
$$V_0'' \neq 0$$
 $\begin{cases} V_0'' > 0 & \text{stable} \\ V_0'' < 0 & \text{unstable} \end{cases}$

What if $V_0'' = 0$? Then check the 1st non-vanishing $V_0^{(n)}$. If n is odd, e.g.,

$$V(q) = \frac{1}{3!} V_0^{\prime\prime\prime} q^3 \qquad F(q) = -\frac{\partial V}{\partial q} = -\frac{1}{2!} V_0^{\prime\prime\prime} q^2 \implies \text{unstable}$$

If m is even and $V_0^{(n)} > 0$, then the particle is stable.

For *n* degrees of freedom, assume $q_1 = q_2 = \dots = q_n = 0$ is the equilibrium point; expand $V(\vec{q})$:

$$V = V(q_1, q_2, ..., q_n) = \frac{1}{2}(k_{11}q_1^2 + k_{12}q_1q_2 + k_{21}q_2q_1 + k_22q_2^2 + ...)$$

where $k_{ij} = \frac{\partial^2 V}{\partial q_i \partial q_j}$. The equation above is a "quadratic form."

 \vec{q} is stable if the quadratic form is positive definite, i.e. V = 0 or V > 0 for all \vec{q} . We may show this is true if

$$k_{11} > 0$$
 , $k_{22} > 0$ and $\begin{vmatrix} k_{11} & k_{12} \\ k_{21} & k_{22} \end{vmatrix} > 0$, and so on

Example: Stability of round-bottomed objects



O is the center of curvature. Note that $h = a - (a - b) \cos \theta$. We have that

$$V = mgh = mga - mg(a - b)\cos\theta$$

 $F = -\frac{\partial V}{\partial \theta} = -mg(a-b)\sin\theta \implies \theta = 0$ is an equilibrium point

$$V_0'' = \frac{\partial^2 V}{\partial \theta^2}\Big|_{\theta=0} = mg(a-b)\cos\theta\Big|_{\theta=0} = mg(a-b) \implies \begin{cases} a > b \implies \text{stable}\\ a < b \implies \text{unstable}\\ a = b \implies \text{neutral} \end{cases}$$

Small Oscillations: Suppose we have stability and a > b. Then for $\theta << 1$

$$V(\theta) \approx \frac{1}{2} V_0'' \theta^2 = \frac{1}{2} mg(a-b)\theta^2$$

T=?. Assume the object rolls without slipping $\implies v_{cm}=b\dot{\theta}$ for $\theta<<1$. Then

$$T=\frac{1}{2}m(b\dot{\theta})^2+\frac{1}{2}I_{cm}\dot{\theta}^2$$

It follows that

$$\mathcal{L} = T - V = \frac{1}{2}mb^2 \left[1 + \frac{I_{cm}}{mb^2} \right] \dot{\theta}^2 - \frac{1}{2}mg(a-b)\theta^2$$
$$\frac{d}{dt} \left(\frac{\partial \mathcal{L}}{\partial \dot{\theta}} \right) = \frac{\partial \mathcal{L}}{\partial \theta} \implies mb^2 \left(1 + \frac{I_{cm}}{mb^2} \right) \ddot{\theta} = -mg(a-b)\theta$$

We call $mb^2(1 + \frac{I_{cm}}{mb^2})$ the m_{eff} . Recall that $mg(a - b) = V_0''$. Our differential equation becomes

$$\ddot{\theta} + \frac{V_0''}{m_{eff}}\theta = 0$$

It follows that $\omega_{osc} = \sqrt{V_0''/m_{eff}}$ is the frequency of "small oscillation."

Coupled Harmonic Oscillators: Normal Modes or Collective Normal Modes of Oscillation



(i): Suppose $x_1(t = 0) > 0$ and $x_2(0) = 0$. m_1 will oscillate, and since it is coupled to the second mass, part of its energy will be transferred to m_2 . Energy will flow back and forth! Each individual mass cannot oscillate alone!

(ii): Consider displacing $x_1 = x_2 = x$. The spring K' is not compressed/stretched, hence the restoring force is 2Kx. Since the total mass is 2m, the frequency is

$$\omega_1 = \sqrt{\frac{2K}{2m}} = \sqrt{\frac{K}{m}}$$



This is called a "normal mode" or "collective mode."

(iii): Another initial condition where $x_1 = -x_2$ is called "breathing mode." In this situation, the center of the middle spring remains fixed while m_1 and m_2 move in and out together.



Find frequency? Restoring force on *each* mass is $F = -kx_1 - 2K'x_1 = -(K + 2K')x$. Note that 2K is the K_{eff} of the middle spring cut in half (we are exploiting symmetry here). Hence

$$\omega_2 = \sqrt{\frac{K + 2K'}{m}}$$

All possible motions are linear combinations of ω_1 and ω_2 .

Method of Solution

$$\mathcal{L} = \frac{1}{2}m\dot{x}_1^2 + \frac{1}{2}m\dot{x}_2^2 - \frac{1}{2}Kx_1^2 - \frac{1}{2}K'(x_1 - x_2)^2 - \frac{1}{2}kx_2^2$$

Using $\frac{d}{dt}(\frac{\partial \mathcal{L}}{\partial \dot{q}}) = \frac{\partial \mathcal{L}}{\partial q}$ gives us

$$m\ddot{x}_1 = -Kx_1 - K'(x_1 - x_2)$$

$$m\ddot{x}_2 = K'(x_1 - x_2) - Kx_2$$

$$\begin{pmatrix} m & 0 \\ 0 & m \end{pmatrix} \begin{pmatrix} x_1 \\ x_2 \end{pmatrix} = - \begin{pmatrix} +(K+K') & -K \\ -K' & (K+K') \end{pmatrix} \begin{pmatrix} x_1 \\ x_2 \end{pmatrix}$$

or, in matrix form, $\mathbb{M} \cdot \ddot{\vec{q}} = -\mathbb{K} \cdot \vec{q}$. We search for "normal mode solutions" of the form $\vec{q} = \vec{a} \cos(\omega t - \delta)$. Since $\ddot{\vec{q}} = -\omega^2 \vec{q}$:

$$-\omega^2 \mathbb{M} \cdot \vec{q} = -\mathbb{K} \cdot q \quad , \quad \text{divide by} \cos(\omega t - \delta)$$
$$\implies \quad (\mathbb{K} - \omega^2 \mathbb{M}) \cdot \vec{a} = 0$$

This is a homogeneous system, non-zero solutions exist only when $det(\mathbb{K} - \omega^2 \mathbb{M}) = 0$. If \mathbb{M} is diagonal, our problem is to find the eigenvalues ω^2 and eigenvectors \vec{a} of \mathbb{K} .

$$\begin{vmatrix} (K+K') - m\omega^2 & -K' \\ -K' & (K+K') - m\omega^2 \end{vmatrix} = 0 \implies [(K+K') - m\omega^2]^2 - K'^2 = 0$$
$$(K+K') - m\omega^2 = \pm K \implies \omega^2 = \frac{K+K' \pm K'}{m} = \begin{cases} \omega_- = \omega_1 = \frac{k}{m} \\ \omega_+ = \omega_2 = \frac{2K'+K}{m} \end{cases}$$

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Eigenvectors \vec{a} ? Plug $w_1^2 = k/m$:

$$\begin{pmatrix} (K+K')-m\omega_1^2 & -K'\\ -K' & (K+K')-m\omega_1^2 \end{pmatrix} \begin{pmatrix} a_{11}\\ a_{21} \end{pmatrix} = \begin{pmatrix} 0\\ 0 \end{pmatrix}$$
$$\implies K' \begin{pmatrix} 1 & -1\\ -1 & 1 \end{pmatrix} \begin{pmatrix} a_{11}\\ a_{21} \end{pmatrix} = \begin{pmatrix} 0\\ 0 \end{pmatrix}$$
$$\implies a_{11}-a_{21}=0 \implies a_{11}=a_{21} \implies \vec{a} = \begin{pmatrix} 1\\ 1 \end{pmatrix} A$$

Similarly with $\omega_2^2 = (2K' + K)/m$ we get

$$\begin{pmatrix} (K+K') - (2K'+K)' & -K' \\ -K' & (K+K') - (2K'+K) \end{pmatrix} \begin{pmatrix} a_{11} \\ a_{21} \end{pmatrix} = \begin{pmatrix} 0 \\ 0 \end{pmatrix}$$
$$\implies K' \begin{pmatrix} 1 & 1 \\ 1 & 1 \end{pmatrix} \begin{pmatrix} a_{11} \\ a_{22} \end{pmatrix} = \begin{pmatrix} 0 \\ 0 \end{pmatrix}$$
$$\implies a_{11} = -a_{22} \implies \vec{a} = \begin{pmatrix} 1 \\ 1 \end{pmatrix} B$$

Therefore the general solution is given by

$$\vec{q} = \begin{pmatrix} x_1 \\ x_2 \end{pmatrix} = A\cos(\omega_1 t + \delta_1) \begin{pmatrix} 1 \\ 1 \end{pmatrix} + B\cos(\omega_2 t + \delta_2) \begin{pmatrix} 1 \\ 1 \end{pmatrix}$$

Two Normal Modes: 4 constants of motion, determined by the initial conditions $x_1(0), \dot{x}_1(0), x_2(0), \dot{x}_2(0)$

Another simple way to solve the problem is to transform to *normal coordinates*:

$$\begin{cases} \ddot{x}_1 + (\frac{K+K'}{m})x_1 - (\frac{K'}{m})x_2 = 0 & (1) \\ \ddot{x}_2 + (\frac{K+K'}{m})x_2 - (\frac{K'}{m})x_1 = 0 & (2) \end{cases}$$

Add and subtract:

$$(1) + (2) \implies (x_1 + x_2) + \left[\left(\frac{K + K'}{m}\right) - \frac{K'}{m}\right](x_1 + x_2) = 0$$

$$(1) - (2) \implies (x_1 + x_2) + \left[\left(\frac{K + K'}{m}\right) + \frac{K'}{m}\right](x_1 + x_2) = 0$$

Therefore when transforming to $Q_1 = x_1 + x_2$ and $Q_2 = x_1 - x_2$ we get

$$\begin{cases} \ddot{Q}_1 + \frac{K}{m}Q_1 = 0\\ \ddot{Q}_2 + \frac{2K' + K}{m}Q_2 = 0 \end{cases}$$

These are known as decoupled equations.

$$\begin{pmatrix} Q_1 \\ Q_2 \end{pmatrix} = \frac{1}{\sqrt{2}} \begin{pmatrix} 1 & 1 \\ 1 & -1 \end{pmatrix} \begin{pmatrix} x_1 \\ x_2 \end{pmatrix} \implies \vec{x} = \mathbb{O} \cdot \vec{Q}$$

Actually we may show that this transform matrix equals

$$\mathbb{O} = (\vec{a}_1 \vec{a}_2) = \begin{pmatrix} a_{11} & a_{12} \\ a_{21} & a_{22} \end{pmatrix}$$

Proof: $\vec{a}_1 = \mathbb{O} \cdot \begin{pmatrix} 1 \\ 0 \end{pmatrix}$ and $\vec{a}_2 = \mathbb{O} \cdot \begin{pmatrix} 0 \\ 1 \end{pmatrix}$ where

$$\mathbb{M}^{-1} \cdot \mathbb{K} \cdot \vec{a}_1 = \frac{k}{m} \vec{a}_1 \implies \mathbb{M}^{-1} \cdot \mathbb{K} \cdot \mathbb{O} \cdot \binom{1}{0} = \frac{k}{m} \mathbb{O} \cdot \binom{1}{0}$$

$$\implies (1 \ 0)(\mathbb{O}^{-1}(\mathbb{M} \cdot \mathbb{K})\mathbb{O})(^1_0) = \frac{k}{m}$$
$$\implies (0 \ 1)(\mathbb{O}^{-1}(\mathbb{M} \cdot \mathbb{K})\mathbb{O})(^1_0) = 0$$

The same goes for \vec{a}_2 . Therefore $\mathbb{O} = (\vec{a}_1 \ \vec{a}_2)$

Now

$$\mathbb{M} \cdot \ddot{\vec{q}} = -\mathbb{K} \cdot \vec{q} \implies \ddot{\vec{q}} = -\mathbb{M}^{-1} \cdot \mathbb{K} \cdot \vec{q}$$
$$\implies (\mathbb{O}^{-1} \cdot \vec{q}) = -\mathbb{O}^{-1} \cdot (\mathbb{M}^{-1} \cdot \mathbb{K}) \cdot \mathbb{O} \cdot (\mathbb{O}^{-1} \cdot \vec{q})$$
$$\implies \begin{pmatrix} \ddot{Q}_1 \\ \ddot{Q}_2 \end{pmatrix} = -\begin{pmatrix} k/m & 0 \\ 0 & (2K'+K)/m \end{pmatrix} \cdot \begin{pmatrix} Q_1 \\ Q_2 \end{pmatrix}$$

The reference system of the normal coordinates "decouples" the problem.